

Predictive design and layer-by-layer characterization of 3D-printed multi-layer micro-perforated panels for broadband sound absorption

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Introduction

Environmental noise pollution is a major public health issue, particularly in tropical island environments such as La Réunion, where noise is compounded by humidity constraints that degrade conventional fibrous and porous absorbers over time. Long-term exposure contributes to sleep disturbance, cognitive impairment and cardiovascular diseases [1]. Conventional sound-absorbing materials such as mineral wool or fiberglass require significant thickness to be effective at low frequencies and are particularly unsuitable for tropical climates due to moisture-induced degradation [2-3].

Micro-Perforated Panels (MPPs) offer a promising sustainable alternative: lightweight, tunable broadband absorbers requiring no fibrous filling, fabricable from recyclable thermoplastics via FDM 3D printing. A Transfer Matrix Method (TMM) simulator was developed in Python to predict $\alpha(f)$ and $Z(f)$ of multi-layer configurations. This work presents the design, simulation and experimental characterization of multi-layer FDM-printed MPP absorbers, and proposes a calibration methodology to bridge the gap between TMM predictions and measurements.

THEORETICAL BACKGROUND

A micro-perforated panel absorber (MPPA) consists of a thin panel of thickness t , perforated with holes of diameter d , placed before a rigid-backed air cavity of depth D [4-5]. At normal incidence, the normalized surface impedance is:

$$Z_1 = Z_{MPP} + Z_D$$

where Z_{MPP} is the panel impedance and $Z_D = -jZ_0 \cot(\omega D/c_0)$ is the cavity impedance. The normal-incidence absorption coefficient is:

$$\alpha = 1 - \left| \frac{Z_1 - 1}{Z_1 + 1} \right|^2$$

Maximum absorption ($\alpha \rightarrow 1$) occurs at resonance frequency:

$$f_0 \approx \frac{c_0}{2\pi} \sqrt{\frac{\phi}{L_{eff} D}}$$

where c_0 is the speed of sound, ϕ the perforation ratio, and $L_{eff} = t + \delta$ the effective perforation length including the end-correction δ accounting for air mass radiation at hole openings.

Experimental setup

Measurements were performed in a circular impedance tube (\varnothing 40 mm) using the transfer function method (ISO 10534-2) [6] over 50–4500 Hz at $\sim 23^\circ\text{C}$. Three three-layer MPP assemblies (Fig 1-3) were tested, sharing identical cavity depths ($D = 20$ mm) and hexagonal perforation pattern but differing in hole diameter d and centre-to-centre spacing a . A solid PLA cylinder confirmed quasi-rigid backing behaviour ($\alpha < 0.21$ across the full range), validating the rigid termination assumption of the TMM.

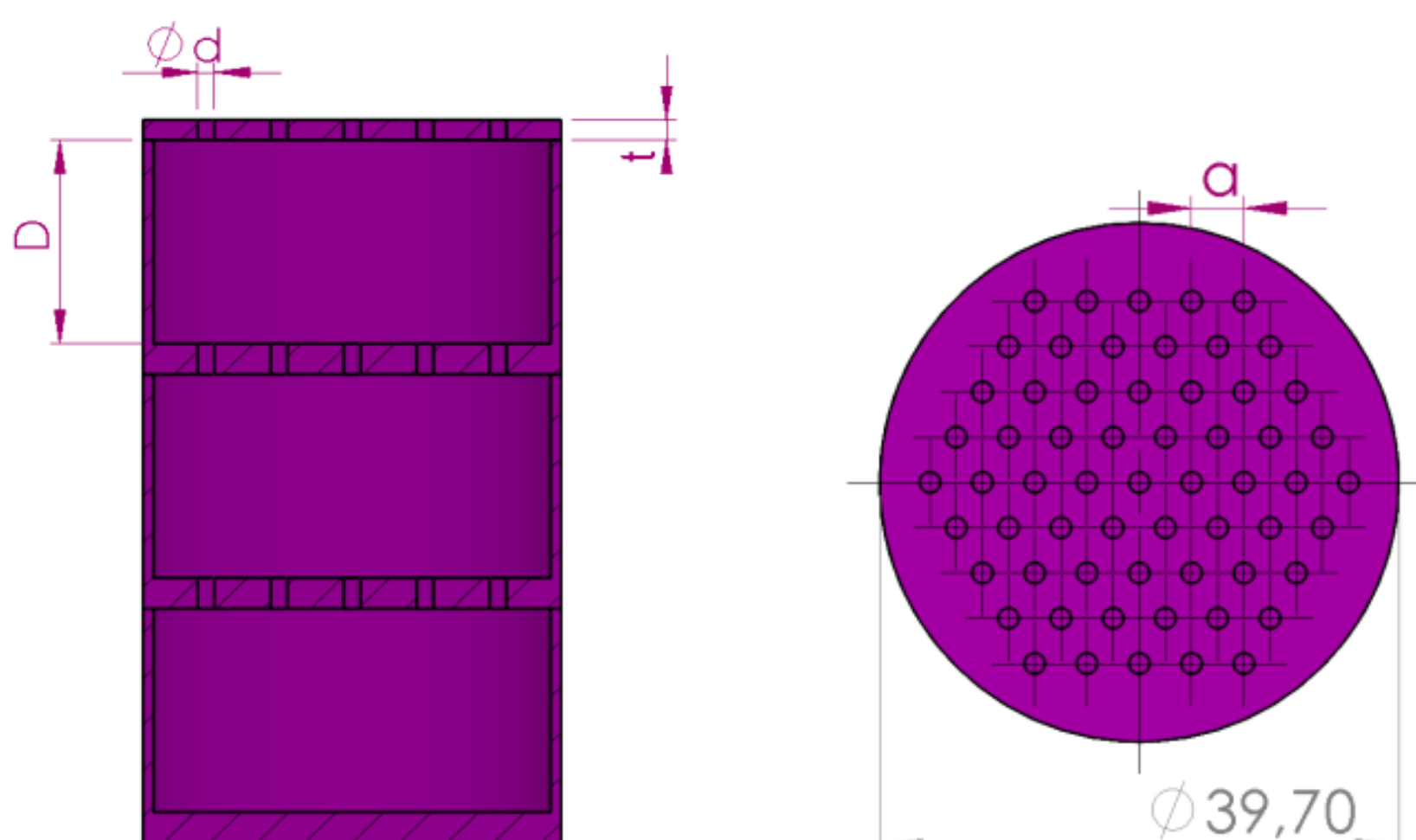


Fig. 1. Multi-layer MPP specimens: CAD design

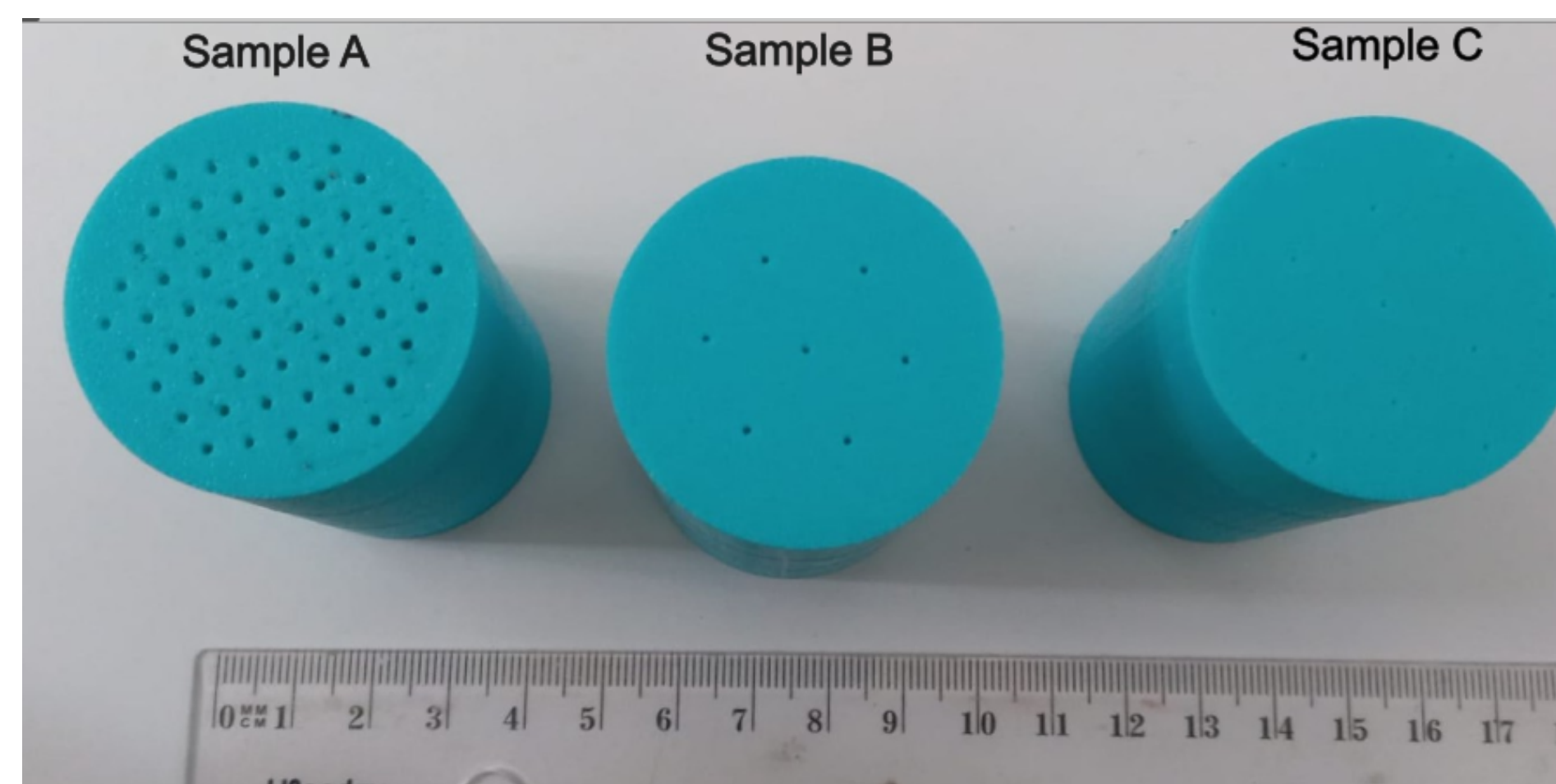


Fig. 2. Multi-layer MPP specimens: FDM-printed PLA samples (sample A to C from left to right).

Tab. 1. Summary of fabricated specimens and key geometric parameters. d : hole diameter; a : distance between the center of two holes; t : panel thickness; ϕ : perforation ratio; D : cavity depth; n : number of holes.

| Samples | Config. | d (mm) | a (mm) | t (mm) | ϕ (%) | D (mm) | n |
|----------|---------|----------|----------|----------|------------|----------|-----|
| Sample A | Layer 1 | 1.60 | 4 | 2 | 9.91 | 20 | 61 |
| | Layer 2 | 1.60 | 4 | 3 | 9.91 | 20 | 61 |
| | Layer 3 | 1.60 | 4 | 3 | 9.91 | 20 | 61 |
| Sample B | Layer 1 | 1.20 | 10 | 2 | 0.68 | 20 | 7 |
| | Layer 2 | 1.20 | 10 | 3 | 0.68 | 20 | 7 |
| | Layer 3 | 1.20 | 10 | 3 | 0.68 | 20 | 7 |
| Sample C | Layer 1 | 1 | 10 | 2 | 0.83 | 20 | 13 |
| | Layer 2 | 1 | 10 | 3 | 0.83 | 20 | 13 |
| | Layer 3 | 1 | 10 | 3 | 0.83 | 20 | 13 |

Results

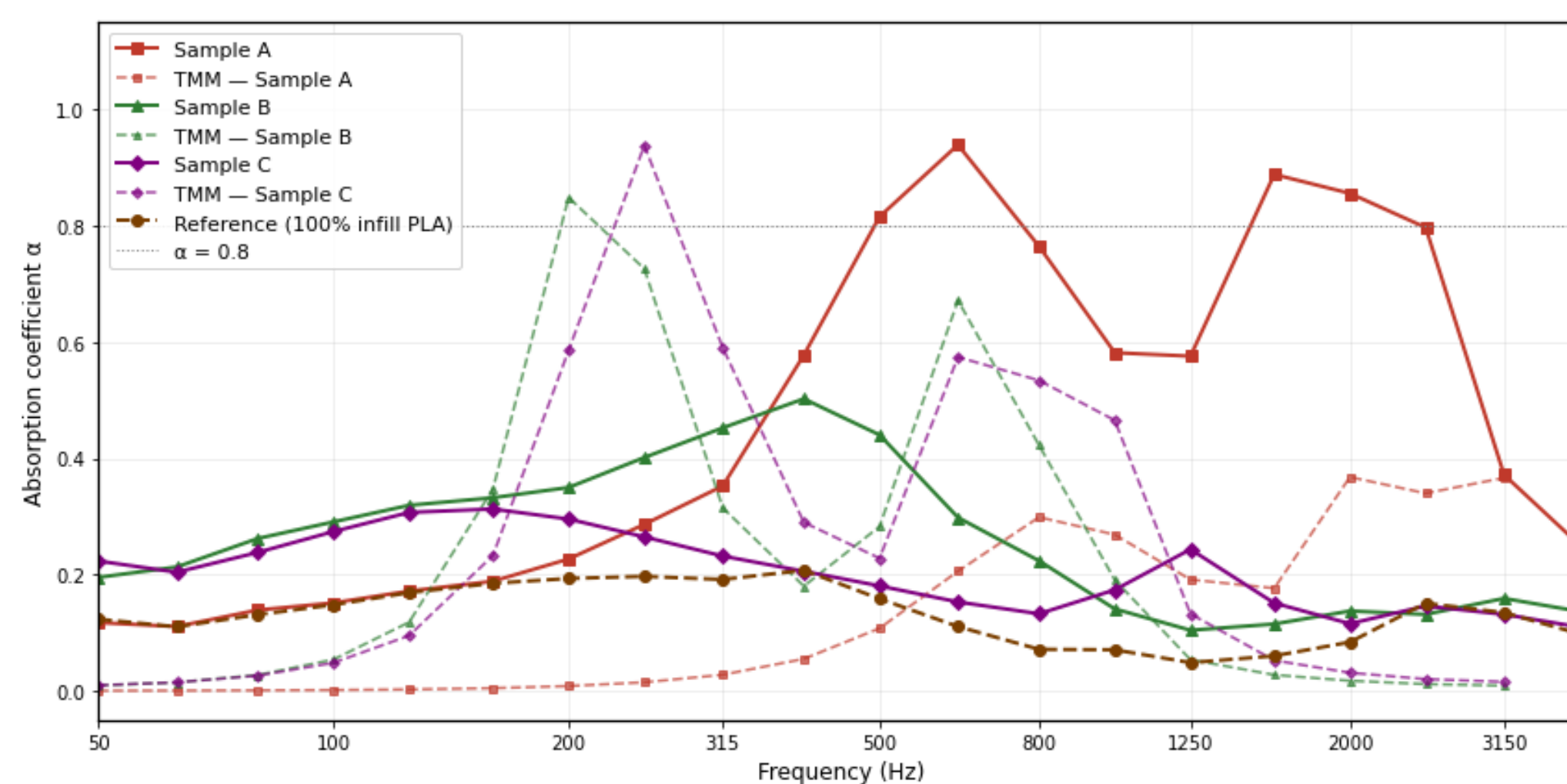


Fig. 3. Measured absorption coefficient (third-octave bands) for samples A, B and C and the PLA reference cylinder. TMM predictions (dashed lines) are shown for each sample

Sample A achieves $\alpha > 0.8$ in two frequency bands: 500–630 Hz ($\alpha_{max} = 0.94$) and 1600–2000 Hz. The TMM significantly underestimates absorption ($\alpha_{max} \approx 0.4$), attributable to FDM wall roughness increasing viscous resistance by a factor of 10–15.

Sample B shows a moderate peak ($\alpha_{max} = 0.50$ at 400 Hz). The TMM overestimates, predicting $\alpha \approx 0.93$ at 250 Hz.

Sample C exhibits a flat, featureless curve ($\alpha_{max} = 0.31$), FDM surface irregularities fully suppress the Helmholtz resonance mechanism at this hole diameter.

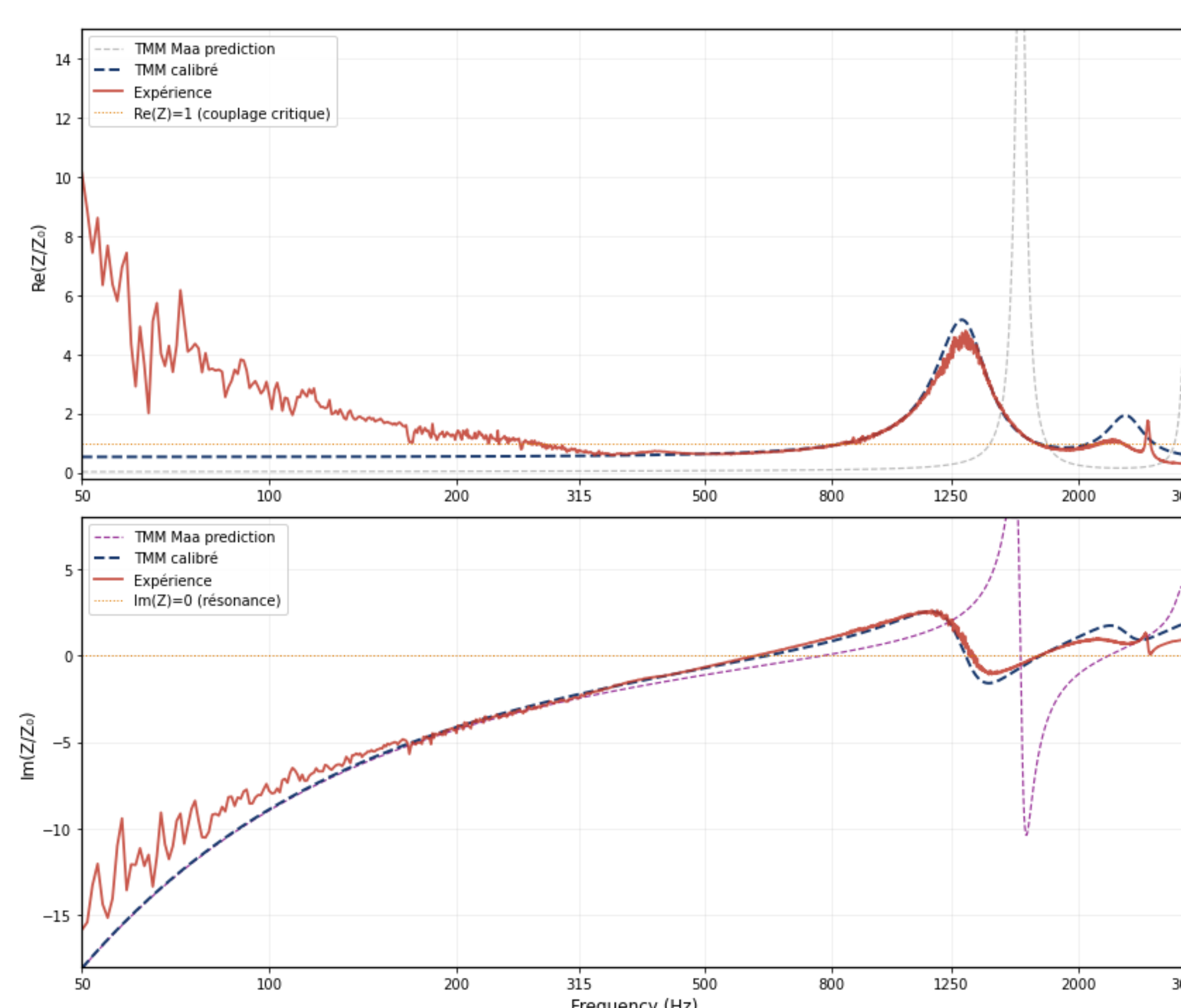


Fig. 4. Normalized impedance $Z(f)$ of sample A: real part $\text{Re}(Z)$ (top) and imaginary part $\text{Im}(Z)$ (bottom). TMM nominal (purple), experiment (red) and calibrated TMM (blue)

Impedance analysis of Sample A reveals that $\text{Im}(Z)$ is well predicted up to 1300 Hz, while $\text{Re}(Z)$ is underestimated by a factor of 10–15. A resonance frequency shift of 21.8% (774 Hz predicted vs. 605 Hz measured) indicates that the effective end-correction significantly exceeds the Fok prediction.

An empirical calibration using a rugosity factor ξ_r and end-correction factor ξ_δ substantially reduces the RMSE for Sample A ($\text{Re}(Z)$: 3.13 \rightarrow 0.72; $\text{Im}(Z)$: 3.00 \rightarrow 0.57) but fails for Sample C, revealing a fundamental acoustic regime change at small hole diameters.

Tab. 2. Calibration results. RMSE computed over 50–3000 Hz on normalized $\text{Re}(Z)$ and $\text{Im}(Z)$.

| Samples | Calib. | ξ_r | ξ_δ | R_0 | RMSE $\text{Re}(Z)$ | RMSE $\text{Im}(Z)$ |
|----------|--------------|---------|--------------|-------|---------------------|---------------------|
| Sample A | Pur Maa | 1 | 1 | 1 | 3.13 | 3.0 |
| | Calib.(auto) | 5.14 | 3.80 | 0 | 0.72 | 0.57 |
| Sample B | Pur Maa | 1 | 1 | 1 | 3.54 | 3.80 |
| | Calib.(auto) | 2.37 | 0.8 | 0 | 2.00 | 2.99 |
| Sample C | Pur Maa | 1 | 1 | 1 | 3.51 | 7.50 |
| | Calib.(auto) | 0.5 | 5 | 0 | 3.66 | 8.56 |

These findings establish a clear hierarchy of FDM-induced effects: wall roughness dominates the resistive component $\text{Re}(Z)$, while the reactive component $\text{Im}(Z)$ remains well captured by the smooth-wall Maa model up to 1300 Hz. This separation is practically valuable, it means TMM can still reliably guide resonance frequency tuning through cavity depth D and perforation ratio ϕ , while absorption amplitude requires a single empirical $\text{Re}(Z)$ measurement for calibration.

Conclusions

Multi-layer FDM-printed MPP absorbers demonstrate strong broadband absorption potential: Sample A achieves $\alpha_{max} = 0.94$, significantly exceeding TMM predictions. Impedance analysis identifies FDM wall roughness as the dominant factor inflating $\text{Re}(Z)$, while $\text{Im}(Z)$ remains well captured by the Maa model up to 1300 Hz. This separation provides a practical diagnostic framework: TMM retains validity for resonance frequency tuning via D and ϕ , but requires empirical $\text{Re}(Z)$ calibration for accurate amplitude prediction. Future work will target frequency-specific multi-layer configurations and systematic characterization of printing parameter effects on Z_{eff_MPP} , toward reliable predictive design of sustainable FDM-printed acoustic metamaterials.

These results confirm that FDM printing introduces systematic deviations from classical MPP theory, primarily through wall roughness and dimensional shrinkage [7], but does not fundamentally prevent the design of efficient broadband absorbers. The proposed calibration framework provides a practical route toward reliable predictive design, paving the way for sustainable, low-cost acoustic treatments adapted to tropical environments.

References

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